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# Design of Heat Recovery System in a Sheet Rock Plant

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#### ABSTRCT

This report is about the design of heat recovery system in a sheet rock plant. A sheet rock is basically a gypsum board which is mainly used as finish for walls, ceilings in households and commercial buildings. According to the given problem, during summer months the city water supplies to the manufacturing plant at a higher temperature of 90° F but during other months the temperature of water supply as 45° F. The demand of water temperature is 85° F for the process so the city water is heated by natural gas burners when it's in a storage tank. During the final stage of sheet rock production heated air is used in a closed oven with sheet rock for the drying process. The air gets exhausted from aluminium stacks. This waste heat energy is being recovered it in the form of energy with the help of specially selected heat exchanger. The energy recovered will be minimising the need of natural gas burner which is to heat the city water receiving at a temperature of 45° F. This would reduce the need of natural gas to use as the main source of heating the city water for production of sheet rock. The heat recovery system would be compatible to work for 24 hr in a 6 days week frame till 8 months. The report will include the type of heat exchanger selected, pump for heat recovery and piping system with fittings and routing for heat recovery. This would also include the cost involved in the whole additional process as installation, operational and maintenance cost with a calculated payback period of time for investment.

# Nomenclature

•	Density	ρ
•	Specific Heat of water	$Cp_{w}$
•	Specific Heat of flue gas	Cpf
•	Heat Energy (Flue gas)	$q_{\rm f}$
•	Heat Energy (Water)	$q_{\rm w}$
•	Effectiveness	E
•	Mass flow rate (Flue gas)	$m_{\rm f}$
•	Mass flow rate (Water)	$m_{\rm w}$
•	Heat Capacity (Flue gas)	$C_{\mathrm{F}}$
•	Heat Capacity (Water)	$C_{\rm w}$
•	Capacity Ratio	$C_{r}$
•	Temperature Inlet (Flue gas)	$T_{i}$
•	Temperature Inlet (Water)	ti
•	Temperature Outlet (Flue gas)	To
•	Temperature Outlet (Water)	$t_{\rm o}$

•	Log Mean Temperature Difference	$\Delta T_{lm}$
•	Pressure	p
•	Volume	V
•	Length	L
•	Diameter (Pipe)	D
•	Area	A
•	Flow rate	Q
•	Velocity of Flow	V
•	Overall Heat Transfer Coefficient	U

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#### INTRODUCTION

A heat recovery is a mechanical process in which heat is recovered from different sources of a plant to gain the potential of enhancing the energy of the system. The heat recovery system starts from a specially constructed heating surface through rectangular ducts into a cross flow heat exchanger to gain the energy for heating purposes. This hot flue gas, when passed through large no. of tubes in the heat exchanger, will exchange a lot of energy to the crossed flowing water in the tubes. The flowing water is coming at a temperature of 45°F (7.22°C) from the city water supply lines provided by the district water supply unit. The water has headers at both the ends of a heat exchanger device and through pipes, in a half circular pattern, the water will flow to gain heat for the system. The efficiency of hot flue gas transferred into energy form is not 100% due to heat losses at certain junctions and duct fittings. The amount of heat left in the system is moved further towards the stack at a dimension of 34 feet and 50 feet from the ground level. To let the gas exhaust through the stacks at each cross-section, manual dampers are installed in both the stacks as well as heat recovery system inlet and out of the ducting arrangement which would open the surface free for the exhaust when the flue gas was allowed to pass through heat recovery system and mean time both stacks dampers are in closed condition so that total flue gas can be passed through heat recovery system.

As the medium is flue gas and low temperature water hence the heat exchanger has been selected – crossed flow Economiser heat exchanger system which consists tubing arrangement with both end water heaters. The inlet water of 45°F entered in the Economiser header and passed through tubes and return to the heat exchanger out let header and carried away heat energy from the passing cross flow flue gas of 230°F. Assuming the dew point of flue gas 140°F we have calculated the heating surface area of heat exchanger and 140oF flue gas exhausted out of the stack and the heat energy utilised during the process.

After the heat transfer to the water tubes, the water temperature rises from given city water temperature of 45°F (7.22°C) to a temperature of 62.4 °F (16.77°C). The heated water will travel through the MS C class piping of 100 NB length 295 feet towards the water holding tank. The water holding tank has two incoming water inlets. The first one is city water 45°F and another one is heated water of 62.4°F coming through the heat recovery process. This heated water contains an enormous amount of energy to partially fulfil the demand of heat provided by the natural gas burners. Natural gas would now be used partially as a major source of heating for the system of manufacturing a sheet rock in a plant. The total profit from this heat recovery system has been calculated with a consideration of working days 8 months, 6 working days in a week and 24 hours of working.

## **Benefits of Waste Heat Recovery**

Benefits of 'waste heat recovery' can be broadly classified into two categories:

#### **Direct Benefits**

Recovery of waste heat energy has a direct effect on the saving of natural gas which is being used to heat the process water from 45°F to 62.4 °F for 8 months resulting to decrease the operation cost of the process which is reflected in process costing.

#### **Indirect Benefits**

- a) **Reduction in pollution**: By installing heat recovery system the natural gas consumption has reduced which is directly polluting the atmospherere due to higher emission level ie reduced the pollution level.
- b) **Reduction in equipment sizes:** Waste heat recovery reduces the fuel consumption, which leads to reduction in natural gas consumption causing reduction in equipment sizes of all natural gas handling equipments like burners, storage of natural gas and piping etc.
- c) **Auxiliary energy consumption**: Reduction in equipment sizes attracts additional benefits in the form of a reduction in auxiliary energy consumption.

## **Information**

**Given:** The given information to design a project are:

- 1. City water Temperature during 8 months ( ${}^{0}F$ ):  $45{}^{0}F = 7.22{}^{0}C$
- 2. Availability of city water for summer (4 months)  $90^{0}\text{F} = 32.22^{0}\text{C}$
- 3. Process Water requirement: 85°F
- 4. Flow rate of City water: 70 GPM = 19093.9 Kg/Hr (1 GPM = 272.77 Kg/Hr)
- 5. City water line pressure (psig):  $55 \text{ psig} = 3.79 \text{ Kg/CM}^2$  (1 kg/cm<sup>2</sup>= 14.53 Psig)
- 6. Total Flow rate of Flue Gas through both Chimney (CFM): 2\*14000 = 28000 CFM = 16520 m<sup>3</sup>/hr (1 CFM= 0.59 M3/Hour)
- 7. ID of Chimney: 3.6 Ft
- 8. Temperature of Flue Gas at outlet of each stack (F):  $230^{\circ}\text{F} = 110^{\circ}\text{C}$
- 9. Height of 1<sup>st</sup> Chimney: 50 Ft (From ground surface)
- 10. Height of 2<sup>nd</sup> Chimney: 34 Ft (From ground surface)
- 11. Distance from water tank to drying oven: 300 Ft
- 12. Specific heat of water as per table A-6 (Incropera 6<sup>th</sup> edition) = 4184 J/Kg.k
- 13. Specific heat of Flue gas as per table = 1068 J/Kg.K (At  $110^{\circ}\text{C}$ )
- 14. Density of Flue gas as per table =  $0.94 \text{ Kg/m}^3$

#### **ASSUMPTIONS**

The values which are assumed based on the requirements of the heat recovery system project are:

- 1) Dew point temperature of Flue gas in each chimney:  $140^{0}$ F =  $(60^{0}$ C)
- 2) Assuming overall heat transfer coefficient as per table C-2
  - a. Flue Gas  $(U_f) = 45 \text{ W/m}^2 \cdot {}^{0}\text{C}$
  - b. Water  $(U_w) = 625 \text{ W/m}^2$ .  ${}^{0}\text{C}$
- 3) Assuming dimension's for flue gas in Heat Exchanger:

Tubes ID: 0.0050 m, Metal thickness: 1.5 mm; Pitch: 20 mm; Length: 12 m

- 4) Assuming width of Heat Exchanger: 4 m
- 5) Assumed GCV (Gross Calorific Value) of Natural Gas: 9600 Kcal/Kg
- 6) Assumed Rate of Natural Gas: CAD \$ 0.13/kg

## **Heat Exchanger**

Heat exchanger design is a multi-step, iterative process made up of the following steps:

- 1. First start with a selection of heat exchanger based on the given mediums i.e air, water, gas.
- 2. Design ways to connect the heat producing surface with the heat exchanger.
- 3. Calculating the log mean temperature difference between the mediums.
- 4. Assuming the overall heat transfer coefficient given for the mediums of flow in a heat exchanger.
- 5. Calculate the area of effectiveness for the incoming heat in the heat exchanger.
- 6. Calculating the heat transfer rate of incoming heat from the heating surfaces to the heat exchanger.
- 7. Calculating the mass flow rate of the heat.
- 8. Calculate the heat capacity ratio of different mediums for analysing the efficiency of the heat exchanger.
- 9. Calculating the effectiveness through NTU method.
- 10. Calculating the no. of tubes of the heat exchanger for the medium to pass through the heat exchanger.
- 11. Calculating the size and width of the heat exchanger.

#### **Heat Exchanger Selection**

Cross Flow Heat Exchanger: The economiser cross flow heat exchanger is the type of heat exchanger process which has been selected for designing the heat recovery system in a sheet rock plant. In this process, fluid as city water would flow through the tubes in a circular pattern from inlet header to outlet header and the heat energy coming from the drying oven as a flue gas would entered from the duct cross sectional area below and above the heat exchanger area in a cross flow pattern. The arrangement of heat exchanger would be one medium flowing in unmixed matter inside the tubes and another one is cross flowing in a mixed manner outside the tubes in the ducting cross sectional area.

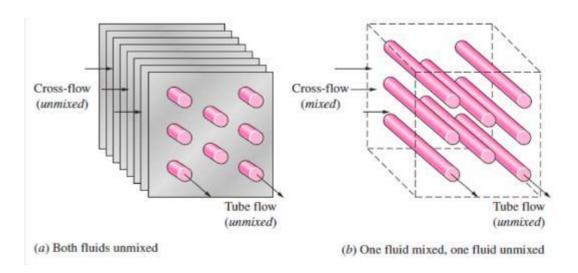


Fig-Cross Flow Heat Exchanger

# Size of Heat Exchanger

# The design of Heat Exchanger Calculations

Heat Exchanger Calculations		
Description	UOM	Values
Given		
Water flow rate	GPM	70
Water flow rate in Kg/Hour (1GPM =272.77 Kg/Hour)	Kg/hour	19093.9
Temperature reqd to be heated up to	oF	85
	oC	29.44
Temperature of city water is being received for 8 months	oF	45
Temperature of city water is being received for 8 months	oC	7.22
Chimney flue gas flow of one chimney	CFM	14000
Chimney flue gas flow of another chimney	CFM	14000
Total Gas flow	CFM	28000
Gas flow ( 1CFM=0.59 M3/hour)	M3/hour	16520.00
Chimney Temperature at outlet be( Ti)	oF	230
Hence Temperature at outlet of flue gas will be ( Ti)	ОС	110
Assuming Due point temperature of flue gas (To)	ОС	60
Density of flue gas as per table	kg/m3	0.94
Specific heat of flue gas as per table	kcal/Kg/oC	0.255

Useful heat energy from flue gas ( qf)	Kcal/hour	m x CPf X(Ti-To) x ρ
Useful heat energy from flue gas ( qf)	Kcal/hour	197992
Heat energy Utilised from chimney flue gas (qf)	Kcal/hour	197992
Mass flow rate of water mCPw	Kg/hour	m x CPw
(assumed specific heat of water =1Kcal/kg/oC)	Kg/hour	19093.9
We know that heat energy from flue gas ( qf) = Heat energy of water (qw)		
hence qf=qw = mCPw x (to-ti)		
ie qf = mCPw x ( to-ti)		
Hence Temperature rise of water (to-ti)	oC	qf/mCPw
Hence Temperature rise of water (to-ti)	oC	10.37
We know that inlet temperature of water is - ti ( as given)	oC	7.22
hence the out let temperature of water will be -to	оС	17.59
Ti	oC	110
to	oC	17.59
То	oC	60
ti	oC	7.22

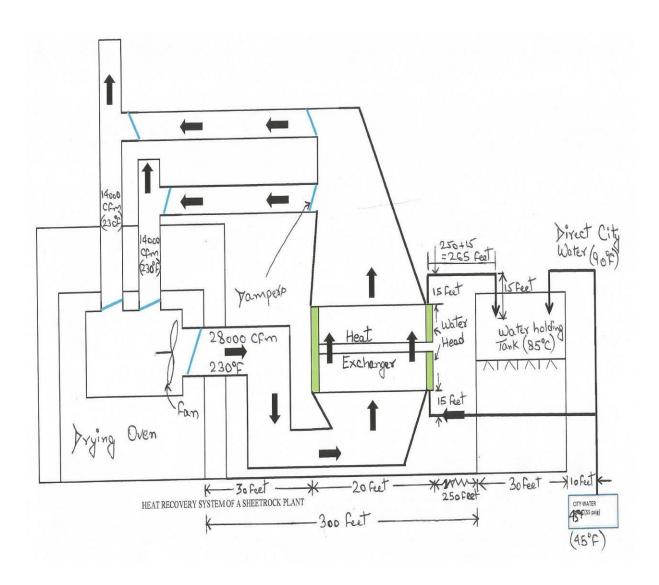
Mass flow rate of water as given (Mc)		Kg/sec	5.30	
Specific heat of water (CPw) as per table A-6		J/kg/k	4184	
Heat capacity of cooling water Cc = Mc x CPc	W/K	J/sec/K	22191	
Mass flow rate of flue gas ( Mf)		Kg/Sec	4.31	
Specific heat of Flue gas as per table A -6		J/kg/k	1068.00	
Heat Capacity of flue gas (Cf) = Mf x CPf	W/K	J/sec/K	4606.88	Cf Max
Heat capacity of Flue gas Cf = Mfx CPf= Heat Capacity of Water Cc x (to-ti)/Ti-To		W/K	4601.23	Cf min
We know that Heat Transfer q max = Cf min x (Ti-ti)		Watt	472904.44	
Actual heat transfer rate is q = Cc x (to-ti)		Watt	230061.62	
We know that Effectiveness E = Heat transfer actual (q)/ Heat Transfer Max (qMax)			0.49	
NTU = -Log ( 1- E(1+CR))/(1+CR) as per graph 11.14 ( INCROPERA 6 th edition)			1.00	
CR = Cmin/Cmax calculated			1.00	

m2	E64*E56/45
m2	102.25
Nos	E72/(3.14*0.005*12)
Nos	543
Nos	E75/2
Nos	271
m	E76*(0.0053+0.002)
m	3.96
m	4
	Nos Nos Nos Nos m

# **Location of Heat Exchanger**

Location of Heat Exchanger -		
volume of chimney exhaust gast	CFM	28000
mass flow rate ( 1CFM =0.59 M3/Hour, 1 hour = 3600 Sec)	m3/sec	E89*0.59/3600
	m3/sec	4.589
density	kg/m3	0.940
mass flow rate	Kg/sec	E91/E92
	Kg/sec	4.882
ID of chimney	Ft	3.6
Inside area of Chimney	Ft2	∏*D2/4
	Ft2	10.17
Velocity of gas flow V= Q/A	FPM	E89/2/E97
	FPM	1376
Hence the velocity of gas flow will be	FPM	1376
	m/s	E100/3.28/60
	m/s	6.99
Duct sizing area for Economizer heat exchanger	Ft2	Q/V
Duct sizing area for Economizer heat exchanger	Ft2	E89/E100
0	Ft2	20.35
Hence the size of duct would be		4.51x 4.51 feet
Hence the location of Heat Exchanger will be away from drying oven	Ft	30
Length of Heat Exchanger is taken	Ft	20
Hence remaining distance from Heat exchanger to Water storage tank	Ft	250

# **Design Layout of the System**



#### **Construction Material**

The construction material used for the cross flow heat exchanger is Medium Carbon Steel. The Carbon steel is a steel alloy which consists of iron and carbon as the main constituent's material. Several other elements such as Manganese, Silicon, and Copper were allowed in carbon steel as low maximum percentages as follows: Manganese (1.65% max), Silicon (0.60% max), Copper (0.60% max). Other elements could be present in the steel in small quantities which won't affect the properties of carbon steel.

There are four types of carbon steel based on the amount of carbon in the alloy as follows:

- 1. **Low Carbon Steel:** Carbon content (0.05-0.25%), Manganese (0.4%) also known as mild steel. Its properties as-Low cost material, easy to shape.
- 2. **Medium Carbon Steel:** Carbon content (0.29-0.54%), Manganese (0.60-1.65%). Its properties as ductile and strong with good wear properties.
- 3. **High Carbon Steel:** Carbon content (0.55-0.95%), Manganese (0.30-0.90%). Its properties as strong and holds shape memory very well ideally for springs and wire.
- 4. **Very High Carbon Steel:** Carbon content (0.96-2.1%). Its high carbon content makes it the extremely strong material, brittle and requires special handling.

# Pump

Pump Calculations		
City water Pressure given	psig	55
Converted (1Kg/cm2=14.53 psig)	Kg/cm2	3.79
Pressure drop across the Heat Exchanger		
e	mm	0.000150
L	m	12
D	m	0.0050
ρ	Kg/m3	999.70
V	m/sec	1.2
Diameter of Tube for Heat Exchanger (D)	m	0.0050
μ Viscosity	N sec/m2	0.00130
Re=ρ x V x D /μ (For Turbulent flow)		4614
F (Friction Factor)		0.038
F=1/SQRT(F) +2*LOG(e/(3.7*D) + 2.51/(Re*SQRT(F)))		
Pressure drop across Heat Exchanger ΔP = ρx F x L /D X V2/2	Kg/m/sec2 ( Pascal	1.87
Frictional loss across Heat Exchanger h= FX(L/D)xV2/2	m2/sec2	66
Head loss across heat exchanger = ΔP /ρ x g +h/g	m	7

Total Suction Head = Net Positive suction head - Head Loss	m	375
Frictional losses in Pipe line ( 200 Feet) h= FX(L/D)xV2/2	m2/sec2	17
Pressure drop across Suction pipe line ΔP = ρx F x L/D X V2/2	Kg/m Sec2 ( Pascal	16798
Head Loss in suction pipe line ( 200 Feet) = ΔP/ρ x g+h/g	m	3.43
Total Discharge head = Static Discharge lift + Discharge head loss + Head Loss across Heat Exchanger	m	214
Static Discharge lift	m	200
Pressure Drop across Heat Exchanger	m	7
Discharge head loss across pipe line	m	7
Head Loss in Discharge pipe line ( 405 Feet) = ΔP /ρ x g +h/g	m2/sec2	7
Frictional losses in Pipe line h= FX(L/D+3Le/D)xV2/2	m2/sec2	35
Pressure drop in discharge pipe line ( 405 Feet) ΔP = ρx F x L /D X V2/2	Kg/m/sec2 ( Pascal	34017
for positive suction Total Dynamic head = Total Discharge Head - Total Suction Head	m	-162
Positive suction Head	Kg/cm2	-1.62
Hence New pump is not required due to available pressure of City water		

## **Piping System**

### **Pipe**

The pipe selected for the flow of city water to and from the heat exchanger is Mild Steel ERW pipes. These pipes are specially designed by galvanising iron (Hot Dip Galvanised) process to avoid any corrosion due to constant contact with water. These pipes are generally used by construction companies and power producing industries.

General Specifications of ERW pipes are:

- 1) Size Range- ½ inch to 6 inch NB
- 2) Thickness range- 6.35 12.7 mm
- 3) Class- A, B, C
- 4) Grade- 1239/I OR BS: 1387 CLASS A, B OR C
- 5) Ends- With Screw & Socket or Plain Ends

## **Pipe fittings**

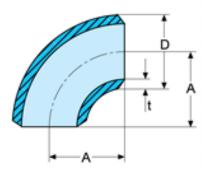
In the heat recovery system, the type of pipe fitting selected is Butt-weld fitting. In particular, to  $90^{0 \text{ elbows}}$ , short radius butt-weld fitting for the flow of water through the heat exchanger. According to the requirement of pipe design, the pairs of butt-weld fitting would be selected in the pipes to flow the water through the heat exchanger into the water holding tank.

There are different forms of butt-weld fittings as:

- 1) Elbow 90<sup>0</sup>, long radius
- 2) Ball Valves
- 3) MS Flanges
- 4) Water Flow meter
- 5) Water inlet and outlet Headers
- 6) Nut and Bolts

# **Dimensions for Pipe Fitting**

The description for butt-weld pipe fitting selected according to the requirement are as follows:



- 1) Nominal Pipe size: 4 inches
- 2) Category: 10 sch (schedules)
- 3) Diameter: 100 mm
- 4) Thickness: 3.05 mm
- 5) Area: 101.6 mm<sup>2</sup>
- 6) Weight (kg/pce): 1.720
- 7) Stainless Steel grade: ASTM A316/316L (Made from welded pipe or plate)

## **Material for Pipe Fitting**

The material selected for butt-weld fitting in the pipe is Austenitic Stainless Steel. This material consists of steel grade- ASTM316/316L which is made from welded pipe or plate.

There are two types of materials used for butt-weld fitting construction as:

- A. Austenitic Stainles Steel: Specifications a r e Stainless steel grades ASTM 304/304L and ASTM 316/316L (dual marked and certified) acc. To ASTM A / ASME SA403, IC acc. To ASTM A262E and PMI tested.
- B. **Duplex and Super Duplex Stainles Steel:** Specifications are Duplex grade UNS S31803 (EN 1.4462 / SAF 2205) and super-duplex grade UNS S32750 (EN 1.4410 / SAF 2507) acc. To ASTM A815, G48A for UNS S32750 and PMI tested.

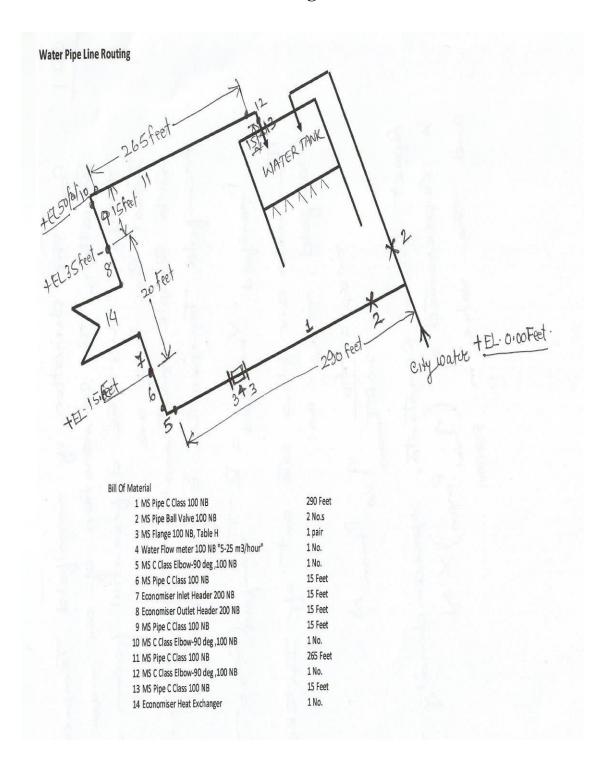
# **Calculations for Pipe Sizing**

Pipe sizing		
Water flow rate	GPM	70
Q	Kg/hour	19093.9
Q	m3/hour	19.094
Q	m3/sec	0.005
Assuming water flow velocity	m/s	1.2
Area required for water	m2	0.00442
We know that A = ∏/4 X D 2		
Hence D2 =4A /∏	m2	0.005630
	mm2	5630
Calculated Diameter of Pipe will be for water transporation	mm	75
Hence the next size of pipe would be	mm	100

The diameter of pipe for water transportation is selected as 100 mm. This is because pipe size is calculated as 75 mm but in actual, pipe size is selected as in the range of  $1-1_{1/2}-1_{1/4}$  inches and 100 mm is relatively equal to 4 inches. It is always desired to take one size higher than the calculated values to minimise the pressure drop across the pipeline considering a reduction in

various losses.

# Routing



## **Construction Material for Pipe**

The construction material selected for the pipe is Mild Steel C Class. The material required for the complete piping system are as follows:

## Piping Material

MS C Class Pipe Size 100 NB Length for inlet of heat exchanger	Feet	305								
MS C Class Pipe Size 100 NB Length for outlet of heat exchanger	Feet	295								
Hence total MS C class Pipe size 100 NB will be	Feet	600								
MS C Class Bends 45 <sup>0</sup>										
Flow Meter Flange mounted Size 100 mm, Flow rate - 5-25										
M <sup>3</sup> /hour	No	1								
MS C Class Flange Table F	Pairs	2								
MS Nut Bolt	M16	16								

### **Economic Calculations**

#### **Input Data**

Natural Gas Rate: CAD \$ 0.13/KG

Flue gas Capacity (2 Chimneys): 435583 Kcal/hour (16520 x 0.255 x 0.94 x 110)

Heat Recovery Capacity (2 Chimneys): - 186346 Kcal/hour

Operating Hours: 4872 hours

#### **Cost of Heat Recovery Calculation**

## Case 1 - (Without Heat Recovery)

Heat Energy required to heat up city water 70 GPM from 45°F (7.22°C) to 85°F (29.42°C)

 $= m \times cp_w \times \rho \times (to-ti) \text{ Kcal/hour}$ 

 $= 70 \times 272.77 \times (29.44-7.22) = 424266 \text{ Kcal/hour}$ 

Assuming Net CV of Natural gas = 9600 Kcal/hour

Hence the Natural Gas consumption would be = 44.19 Kg/hour

Hence the Natural gas consumption for 8 months (4872 hours) == 215294 kg / 8 months

#### Case II - (With Heat Recovery)

Heat Energy required to heat up city water 70 GPM from 45 °F (7.22 °C) to 63.66 °F (17.59 °C)

=  $m \times cpw \times \rho \times (to-ti)$  Kcal/hour

 $= 70 \times 272.77 \times (17.59-7.22) = 197992 \text{ Kcal/hour}$ 

Assuming Net CV of Natural gas = 9600 Kcal/hour

Hence the Natural Gas consumption would be = 20.62 Kg/hour

N a tural gas consumption for 8 months (4872 Hours) == 100481 kg / 8 months

Net Saving in Natural Gas consumption (4872 Hours) == 100481 kg / 8 months

Hence the final natural gas consumption would be = 114813 Kg/8 months

Rate of natural gas == CAD \$ 0.13/KG

Hence net cost saving in 8 months =  $0.13 \times 100481 == CAD \$ 13063.00$ 

**Payback Summary** 

Annual Natural Gas Cost Savings CAD \$ 13063 /year

Heat Recovery Calculations		
Heat Energy utilised to convert water temperature from 45 oF to 63.66 oF	Kcal/hour	197992
Assumed GCV of Natural Gas	Kcal/Kg	9600
Natural Gas saving by heat recovery	Kg/hour	20.62
Time to be calculated ( 203 days x 24 Hours)	hours	4872
Gas qty saving in 8 months	Kg	100481
Assumed Rate of Natual Gas	CAD\$	0.13
Cost saving of Natual gas for 8 months	CAD\$	13063
Installation cost		
Cost of Heat Exchanger	CAD\$	12000
Piping bends, Flanges and Flowmeter	CAD\$	7200
Ducting and dampers	CAD\$	12000
Insulation of inlet ducting and heat exchanger	CAD\$	10000
Total Cost	CAD\$	41200
Contigency @5%	CAD\$	2060
Total Project cost	CAD\$	43260

Operation cost Control of the Contro		
N. J	V1/h	225247
Natural Gas required to heat up water from 63.66 oF to 85 oF for 70 GPM water	Kcal/hour	226317
Qty of Gas required to heat up water as above	Kg/hour	25.86
Qty of Gas required to heat up water as above ( 8 months)	Kg/Year	125990
Operation cost of Natural gas	CAD \$/Year	16379
Man power Cost @ 15 CAD \$/hour	CAD \$/Year	73080
Total operating cost	CAD \$/Year	89458.7
Maintenance Cost		
Stores and spares for Heat Exchanger, Ducting ,Dampers and burners @ 2% Of project cost	CAD \$/Year	865.2
Repair and Maint cost @ 3 % of project cost	CAD \$/Year	1297.8
Total maintenance cost assumed @ 5 % of Project cost per year	CAD \$/Year	2163
Grand Total - ( Project cost + operation cost + Maintenance Cost)	CAD \$/Year	134882
Pay back period ( Project cost/ Annual saving )	Years	3.31

#### **Payback Period for Investment**

From the energy savings and implementation costs calculation, a simple payback period for the system design can be analysed. A simple payback calculates the number of years involved in the investment that would reduce the energy costs by the same amount as the initial cost of implementation.

The simple payback period follows as:

SP = IC / ES

= \$43,260/ \$13,063 per year

= 3.31 years

Where:

SP = Simple Payback Period

IC = Total Implementation Cost for the entire system

ES = Annual Energy Savings expected from the design

This system design is estimated to pay for itself in less than three years and 31 days. This simple payback is calculated based on the initial costs along with the estimated energy consumption reduction and it does not include any types of maintenance costs throughout the system life.

However, the system does not consist of many moving parts leading to the majority of the maintenance costs being associated with the cleaning of the heat exchangers. This maintenance is assumed to be minimal and should not impact the implementation of the system. However, certain maintenance interval estimations have been established and are discussed in previous Sections which can be used to provide a more accurate representation of the payback period.

#### Conclusion

- 1) The heat exchanger type selected is economiser cross flow heat exchanger as:
  - a. Size  $-6 \times 4 \text{ m}^2$
  - b. Length -20 feet (6 m)
  - c. Location -
    - I. Distance of Heat exchanger from drying oven- 30 feet
    - II. The distance of Heat exchanger from the water holding tank-
  - 250 feet d. Material Used Medium Carbon Steel
- Pump Not required as the pressure of water flow is adequate to maintain pressure drop in the water pipeline.
- 3) Piping System as:
  - I. Pipe Size (Nominal Diameter): 100 mm (4 inches)
  - II. Pipe Length from Junction of city water to the Heat exchanger Inlet header: 305 feet
  - III. Pipe Length from Heat Exchanger outlet header to the water holding tank: 295 feet
- 4) Material Used for pipe: Mild Steel C Class
- 5) Material Used for Pipe fitting: Austenitic Stainless Steel (ASTM A 316/316L)
- 6) Total Project Cost Involved: CAD \$ 43260.00
- 7) Payback period: 3.31 years

## **Used Tables**

Table C.2 Representative values of the overall heat transfer coefficients (SI)

Type of Heat Exchanger	U (W/(m² °C))
Water-to-water	850-1700
Water-to-oil	100-350
Water-to-gasoline or kerosene	300-1000
Feedwater heater	1000-8500
Steam-to-light fuel oil	200-400
Steam-to-heavy fuel oil	50-200
Steam condenser	1000-6000
Freon condenser (water cooled)	300-1000
Ammonia condenser (water cooled)	800-1400
Alcohol condenser (water cooled)	250-700
Gas-to-gas	10-40
Water-to-air in finned tubes (water in tubes)	30-60 (air); 400-850 (water)
Steam-to-air in finned tubes (steam in tubes)	30-300 (air); 400-4000 (water)

Source: Çengel, Y.A. (2007) Heat and Mass Transfer: A Practical Approach, 3rd edn, McGraw-Hill, Inc., New York.

t	ρ	Сp	μ *10 <sup>6</sup>	v *10 <sup>6</sup>
[o c]	[kg/m <sup>3</sup> ]	[kJ/kgK]	[Pas]	[m <sup>2</sup> /s]
0	1.295	1.042	15.8	12.2
100	0.95	1.068	20.4	21.54
200	0.748	1.097	24.5	32.8
300	0.617	1.122	28.2	45.81
400	0.525	1.151	31.7	60.38
500	0.457	1.185	34.8	76.3
600	0.405	1.214	37.9	93.61
700	0.363	1.239	40.7	112.1
800	0.33	1.264	43.4	131.8
900	0.301	1.29	45.9	152.5
1000	0.275	1.306	48.4	174.3
1100	0.257	1.323	50.7	197.1
1200	0.24	1.34	53	221

Fig: Flue Gas Property Table

## 894 | Thermodynamics

TABI	LE A-6												
Superl	neated							•					
water													
T	V	и	h	s	V		и	h	S	V	и	h	S
	3.			kJ/k	3					2			kJ/kg
<u>°C</u>	g m <sup>3</sup> /kg	kJ/kg			m <sup>3</sup> /k		kJ/kg		ı kJ/kg	m <sup>3</sup> /kg	kJ/kg	kJ/kg	
	$P_{-}0.01$	MPa (45.8	31°C)*		$P_{-}0.05$	MPa	(81.32°	C)		$P_{-}$	0.10 MPa	a (99.61	°C)
Sat.†	14.670	2437.2	2583.9	8.1488	3	.2403	3 2483	.2 26	645.2	1.6941	2505.6	2675.0	7.3589
50	14.867	2443.3	2592.0				7.593	31					
100	17.196	2515.5	2687.5		2	440-	7 0544	- o	200.4	1.6959			7.3611
150 200	19.513 21.826	2587.9 2661.4	2783.0 2879.6		3	.4187	7 2511 7.695		82.4	1.9367 2.1724			7.6148 7.8356
250	24.136	2736.1	2977.5		3	.8897			780.2	2.4062			8.0346
300	26.446	2812.3	3076.7	9.2827			7.94			2.6389	2810.7		8.2172
400	31.063	2969.3	3280.0		4	.3562			377.8	3.1027			8.5452
500	35.680	3132.9	3489.7		1	.8206	8.159 2735		976.2	3.5655	3132.2		8.8362
600 700	40.296 44.911	3303.3 3480.8		10.1631 10.4056	7	.0200	8.356		770.2	4.0279 4.4900			9.0999 9.3424
800	49.527	3665.4		10.6312	5	.2841	_		75.8	4.9519	3665.0		9.5682
900	54.143	3856.9		10.8429		000	8.538		70.0	5.4137			9.7800
1000	58.758	4055.3		11.0429	6	.2094	4 2968 8.865		279.3	5.8755			9.9800
1100	63.373	4260.0		11.2326	7	.1338			189.3	6.3372			10.1698
1200 1300	67.989 72.604	4470.9 4687.4		11.4132 11.5857	•		9.156		.00.0	6.7988 7.2605			10.3504 10.5229
1000		MPa (120		11.5057		MPa	1 (133.52		706.0		0.40 MPa		
	7 _ 0.20	IVII A (120	.21 0)		_ 0.50	) IVII O	1 (100.02	- 0)		' -	0.40 IVII a	(145.0	1 0)
Sat.	0.88578	2529.1	2706.3				2543.2		.9 6.9917	0.46242			6.8955
150	0.95986	2577.1	2769.1				2571.0		.2 7.0792	0.47088			6.9306
200 250	1.08049 1.19890	2654.6	2870.7 2971.2		0.71 0.79		2651.0 2728.9		.9 7.3132 .9 7.5180	0.53434 0.59520			7.1723 7.3804
300	1.31623	2808.8	3072.1		0.73		2807.0		.6 7.7037	0.65489			7.5677
400	1.54934	2967.2	3277.0		1.03		2966.0		.5 8.0347	0.77265			7.9003
500	1.78142		3487.7				3130.6		.6 8.3271	0.88936			8.1933
600	2.01302		3704.8		1.34		3301.6		.0 8.5915	1.00558			8.4580
700 800	2.24434 2.47550	3479.9 3664.7	3928.8 4159.8		1.49 1.65		3479.5 3664.3		.2 8.8345 .3 9.0605	1.12152 1.23730			8.7012 8.9274
900	2.70656	3856.3	4397.7		1.80		3856.0		.3 9.0003	1.35298			9.1394
1000	2.93755	4054.8	4642.3		1.95		4054.5		.0 9.4726	1.46859			9.3396
1100	3.16848	4259.6	4893.3	9.8497	2.11		4259.4	4893	.1 9.6624	1.58414		4892.9	9.5295
1200	3.39938	4470.5		10.0304	2.26		4470.3		.2 9.8431	1.69966	-		9.7102
1300		4687.1 MPa (151		10.2029	2.42 P 0.60		4686.9 (158.83		.0 10.0157	1.81516	0.80 MPa		9.8828
	F _ 0.30	IVIFA (131	.03 ()	ï	_ 0.00	IVIFA	1 (156.60	, C)		<i>-</i>	U.OU IVIFA	1 (170.4	1 ()
Sat.	0.37483		2748.1				2566.8		.2 6.7593	0.24035	2576.0		6.6616
200	0.42503		2855.8				2639.4		.6 6.9683	0.26088			6.8177
250 300	0.47443 0.52261		2961.0 3064.6				2721.2 2801.4		.6 7.1833 .0 7.3740	0.29321 0.32416			7.0402 7.2345
350	0.57015		3168.1				2881.6		.1 7.5481	0.35442			7.4107
400	0.61731		3272.4				2962.5		.8 7.7097	0.38429			7.5735
500	0.71095		3484.5	8.0893	0.59	200	3128.2	3483	.4 8.0041	0.44332	3126.6	3481.3	7.8692
600	0.80409		3702.5				3299.8		.7 8.2695	0.50186			8.1354
700 800	0.89696 0.98966		3927.0			725 457	3478.1 3663.2		.4 8.5132 .9 8.7395	0.56011			8.3794 8.6061
900	1.08227		4158.4 4396.6		0.82		3855.1		.9 8.7395 .2 8.9518	0.61820 0.67619			8.6061 8.8185
1000	1.17480		4641.4		0.97		4053.8		.1 9.1521	0.73411			9.0189
1100	1.26728	4259.0	4892.6				4258.8		.4 9.3420	0.79197			9.2090
1200			5149.8			309	4469.8		.6 9.5229	0.84980			9.3898
1300	1.45214	4686.6	5412.6	9.7797	1.21	012	4686.4	5412	.5 9.6955	0.90761	4686.1	5412.2	9.5625
				Į.									

TABI	LE A-6													
Super	ntinued)								•					
T			h	_	V		h		V		h			
,	V	и	11	S	V	и	11	S	V	и	11	s kJ/kg		
°C	m <sup>3</sup> /kg	kJ/kg	kJ/kg	kJ/kg K	m <sup>3</sup> /kg	kJ/kg	kJ/kg	kJ/kg K	m <sup>3</sup> /kg	kJ/kg	kJ/kg	KJ/Kg _K		
	Р	1.00 MF	Pa (179.88	C)	F	2 1.20 N	1Pa (187.	96 C)	Р	1.40 MPa	a (195.04	I C)		
Sat.	0.19437	2582.8	2777.1	6.5850	0.16326	2587.8	2783.8	6.5217	0.14078	2591.8	2788.9	<b>6</b> .4675		
200	0.20602	2622.3	2828.3	6.6956	0.16934		2816.1	6.5909	0.14303	2602.7	2803.0	6.4975		
250	0.23275		2943.1	6.9265	0.19241	-	2935.6	6.8313	0.16356	2698.9	2927.9	6.7488		
300	0.25799		3051.6	7.1246	0.21386		3046.3	7.0335	0.18233	2785.7	3040.9	6.9553		
350	0.28250		3158.2	7.3029	0.23455	_	3154.2	7.2139	0.20029	2869.7 2953.1	3150.1	7.1379 7.3046		
400 500	0.30661 0.35411		3264.5 3479.1	7.4670 7.7642	0.25482 0.29464		3261.3 3477.0	7.3793 7.6779	0.21782 0.25216	3121.8	3258.1 3474.8	7.3046 7.6047		
600	0.33411		3698.6	8.0311	0.29464		3697.0	7.0779	0.23210	3295.1	3695.5	7.8730		
700	0.44783		3924.1	8.2755	0.37297		3922.9	8.1904	0.20007	3474.4	3921.7	8.1183		
800	0.49438		4156.1	8.5024	0.41184		4155.2	8.4176	0.35288	3660.3	4154.3	8.3458		
900	0.54083		4394.8	8.7150	0.45059		4394.0	8.6303	0.38614	3852.7	4393.3	8.5587		
1000	0.58721		4640.0	8.9155	0.48928		4639.4	8.8310	0.41933	4051.7	4638.8	8.7595		
1100	0.63354	4257.9	4891.4	9.1057	0.52792	4257.5	4891.0	9.0212	0.45247	4257.0	4890.5	8.9497		
1200	0.67983	4469.0	5148.9	9.2866	0.56652	4468.7	5148.5	9.2022	0.48558	4468.3	5148.1	9.1308		
1300	0.72610	4685.8	5411.9	9.4593	0.60509	4685.5	5411.6	9.3750	0.51866	4685.1	5411.3	9.3036		
									1					
	<i>P</i>	1.60 MF	Pa (201.37	C)	F	2 1.80 N	I <u>P</u> a (207.	11 <u>C</u> )	P 2.00 MPa (212.38 C)					
Sat.	0.12374	2594.8	2792.8	6.4200	0.11037	2597.3	2795.9	6.3775	0.09959	2599.1	2798.3	6.3390		
225	0.13293	2645.1	2857.8	6.5537	0.11678	2637.0	2847.2	6.4825	0.10381	2628.5	2836.1	6.4160		
250	0.14190		2919.9	6.6753	0.12502		2911.7	6.6088	0.11150	2680.3	2903.3	6.5475		
300	0.15866		3035.4	6.8864	0.14025		3029.9	6.8246	0.12551	2773.2	3024.2	6.7684		
350	0.17459		3146.0	7.0713	0.15460		3141.9	7.0120	0.13860	2860.5	3137.7	6.9583		
400	0.19007		3254.9	7.2394	0.16849		3251.6	7.1814	0.15122	2945.9	3248.4	7.1292		
500	0.22029		3472.6	7.5410	0.19551		3470.4	7.4845	0.17568	3116.9	3468.3	7.4337		
600 700	0.24999 0.27941		3693.9 3920.5	7.8101 8.0558	0.22200 0.24822		3692.3	7.7543	0.19962 0.22326	3291.5 3471.7	3690.7 3918.2	7.7043 7.9509		
800	0.27941		4153.4	8.2834	0.24622		3919.4 4152.4	8.0005 8.2284	0.22326	3658.0	4151.5	8.1791		
900	0.33780		4392.6	8.4965	0.30020		4391.9	8.4417	0.27012	3850.9	4391.1	8.3925		
1000	0.36687		4638.2	8.6974	0.32606		4637.6	8.6427	0.29342	4050.2	4637.1	8.5936		
1100	0.39589		4890.0	8.8878	0.35188		4889.6	8.8331	0.31667	4255.7	4889.1	8.7842		
1200	0.42488		5147.7	9.0689	0.37766		5147.3	9.0143	0.33989	4467.2	5147.0	8.9654		
1300	0.45383	4684.8	5410.9	9.2418	0.40341	4684.5	5410.6	9.1872	0.36308	4684.2	5410.3	9.1384		
		0.50 ME	)- (000 OF	<b>C</b> \	_	2 2 20 1	ID- (000	05.0)	P	2 FO MD	(040.50	. 0)		
Cot			<del>223.95</del> 2801.9	_			<u>Pa (233.</u>				<u>242.56</u>			
Sat. 225	0.07995 0.08026		2805.5	6.2558 6.2629	0.06667	2003.2	2803.2	6.1856	0.05706	2603.0	2802.7	0.1244		
250	0.08705		2880.9	6.4107	0.07063	2644 7	2856.5	6.2893	0.05876	2624.0	2829.7	6.1764		
300	0.00703		3009.6	6.6459	0.07003		2994.3	6.5412	0.06845	2738.8	2978.4	6.4484		
350	0.10979		3127.0	6.8424	0.09056		3116.1	6.7450	0.07680	2836.0	3104.9	6.6601		
400	0.12012		3240.1	7.0170	0.09938		3231.7	6.9235	0.08456	2927.2	3223.2	6.8428		
450	0.13015		3351.6	7.1768	0.10789		3344.9	7.0856	0.09198	3016.1	3338.1	7.0074		
500	0.13999	3112.8	3462.8	7.3254	0.11620	3108.6	3457.2	7.2359	0.09919	3104.5	3451.7	7.1593		
600	0.15931	3288.5	3686.8	7.5979	0.13245	3285.5	3682.8	7.5103	0.11325	3282.5	3678.9	7.4357		
700	0.17835	3469.3	3915.2	7.8455	0.14841	3467.0	3912.2	7.7590	0.12702	3464.7	3909.3	7.6855		
800	0.19722		4149.2	8.0744	0.16420		4146.9	7.9885	0.14061	3652.5	4144.6	7.9156		
900	0.21597		4389.3	8.2882	0.17988		4387.5	8.2028	0.15410	3846.4	4385.7	8.1304		
1000	0.23466		4635.6	8.4897	0.19549		4634.2	8.4045	0.16751	4046.4	4632.7	8.3324		
1100	0.25330		4887.9	8.6804	0.21105		4886.7	8.5955	0.18087	4252.5	4885.6	8.5236		
1200	0.27190		5146.0	8.8618	0.22658		5145.1		0.19420	4464.4	5144.1	8.7053		
1300	0.29048	4683.4	5409.5	9.0349	0.24207	4082.6	5408.8	8.9502	0.20750	4681.8	5408.0	8.8786		

T	V	и	h	s	V	и	h	s	V	и	h	S	
°C	m³/kg	kJ/kg	kJ/kg	kJ/kg K	m <sup>3</sup> /kg	kJ/kg	kJ/kg	kJ/kg K	m <sup>3</sup> /kg	kJ/kg	kJ/kg	kJ/kg _K	
		P 4.0 MPa	a (250.35	C)	P 4.5 MPa	(257.44)	C)		P 5.0 MPa (263.94 C)				
Sat.	0.04978	2601.7	2800.8	6.0696	0.04406	2599.7	2798.0	6.0198	0.03945	2597.0		5.9737	
275	0.05461	2668.9	2887.3	6.2312	0.04733	2651.4	2864.4	6.1429	0.04144	2632.3	2839.5	6.0571	
300	0.05887	2726.2	2961.7	6.3639	0.05138	2713.0	2944.2	6.2854	0.04535	2699.0		6.2111	
350	0.06647		3093.3	6.5843	0.05842	2818.6	3081.5	6.5153	0.05197	2809.5		6.4516	
400	0.07343		3214.5	6.7714	0.06477	2914.2	3205.7	6.7071	0.05784	2907.5		6.6483	
450	0.08004		3331.2	6.9386	0.07076	3005.8	3324.2	6.8770	0.06332	3000.6		6.8210	
500	0.08644		3446.0 3674.9	7.0922 7.3706	0.07652 0.08766	3096.0 3276.4	3440.4	7.0323 7.3127	0.06858	3091.8 3273.3		6.9781	
600 700	0.09886 0.11098		3906.3	7.6214	0.08766	3460.0	3670.9 3903.3	7.5647	0.07870 0.08852	3457.7		7.2605 7.5136	
800	0.11098		4142.3	7.8523	0.09830	3648.8	4140.0	7.7962	0.08832	3646.9		7.7458	
900	0.13476		4383.9	8.0675	0.11972	3843.3	4382.1	8.0118	0.10769	3841.8		7.9619	
1000	0.14653		4631.2	8.2698	0.13020	4043.9	4629.8	8.2144	0.11715	4042.6		8.1648	
1100	0.15824		4884.4	8.4612	0.14064	4250.4	4883.2	8.4060	0.12655	4249.3		8.3566	
1200	0.16992	4463.5	5143.2	8.6430	0.15103	4462.6	5142.2	8.5880	0.13592	4461.6		8.5388	
1300	0.18157	4680.9	5407.2	8.8164	0.16140	4680.1	5406.5	8.7616	0.14527	4679.3		8.7124	
		P 6.0 MPa	a (275.59	C)	P 7.0 MPa	(285.83)	C)		P 8.0 MPa	(295.01	C)		
Sat.	0.03245		2784.6	5.8902	0.027378		2772.6	5.8148	0.023525	_		5.7450	
300	0.03619		2885.6	6.0703	0.029492		2839.9	5.9337	0.024279			5.7937	
350	0.04225	2790.4	3043.9	6.3357	0.035262	2770.1	3016.9	6.2305	0.029975	2748.3	2988.1	6.1321	
400	0.04742	2893.7	3178.3	6.5432	0.039958	2879.5	3159.2	6.4502	0.034344	2864.6	3139.4	6.3658	
450	0.05217	2989.9	3302.9	6.7219	0.044187	2979.0	3288.3	6.6353	0.038194	2967.8	3273.3	6.5579	
500	0.05667		3423.1	6.8826	0.048157		3411.4	6.8000	0.041767			6.7266	
550	0.06102		3541.3	7.0308	0.051966	3167.9	3531.6	6.9507	0.045172			6.8800	
600	0.06527		3658.8	7.1693	0.055665		3650.6	7.0910	0.048463			7.0221	
700	0.07355 0.08165		3894.3 4133.1	7.4247 7.6582	0.062850 0.069856		3888.3 4128.5	7.3487 7.5836	0.054829 0.061011	3635.7		7.2822 7.5185	
800 900	0.08163		4376.6	7.8751	0.069656	3835.7	4373.0	7.8014	0.067082			7.5165	
1000	0.00904		4625.4	8.0786	0.076730	4037.5	4622.5	8.0055		4035.0		7.7372	
1100	0.10543		4879.7	8.2709	0.000371		4877.4	8.1982	0.079025			8.1350	
1200	0.11326		5139.4	8.4534	0.097075		5137.4	8.3810	0.084934			8.3181	
1300	0.12107		5404.1	8.6273	0.103781		5402.6	8.5551	0.090817			8.4925	
		P Q O MP:	a (303.35	C)	P 10.0 MF	ر ام (311 مر	) (C)		P 12.5 M	Pa (327.8	1 C)		
Sat.	0.020489		2742.9	5.6791	0.018028		2725.5	5.6159	0.013496			5.4638	
325	0.023284		2857.1	5.8738	0.019877		2810.3	5.7596	0.0.0.00				
350	0.025816		2957.3	6.0380	0.022440	-	2924.0	5.9460	0.016138	2624.9	2826.6	5.7130	
400	0.029960	2849.2	3118.8	6.2876	0.026436	2833.1	3097.5	6.2141	0.020030	2789.6	3040.0	6.0433	
450	0.033524	2956.3	3258.0	6.4872	0.029782	2944.5	3242.4	6.4219	0.023019	2913.7	3201.5	6.2749	
500	0.036793	3056.3	3387.4	6.6603	0.032811	3047.0	3375.1	6.5995	0.025630	3023.2	3343.6	6.4651	
550	0.039885		3512.0	6.8164	0.035655		3502.0	6.7585	0.028033			6.6317	
600	0.042861		3634.1	6.9605	0.038378		3625.8	6.9045	0.030306			6.7828	
650	0.045755		3755.2	7.0954	0.041018		3748.1	7.0408	0.032491			6.9227	
700	0.048589		3876.1	7.2229	0.043597		3870.0	7.1693	0.034612			7.0540	
800	0.054132		4119.2	7.4606	0.048629		4114.5	7.4085	0.038724			7.2967	
900	0.059562		4365.7	7.6802	0.053547		4362.0	7.6290	0.042720 0.046641			7.5195	
	0.064919		4616.7	7.8855	0.058391 0.063183		4613.8	7.8349 8.0289	0.046641			7.7269	
	0.075492		4872.7 5133.6	8.0791 8.2625	0.063163		4870.3 5131.7	8.2126	0.050310			7.9220 8.1065	
	0.073492		5399.5	8.4371	0.007938		5398.0	8.3874	0.054342			8.2819	
	3.220.00	20											
								ı					
		=					=						

TAF	BLE A-6											
	erheated w	ater (Co.	ncluded)									
<u> </u>	V	и	h	s	V	и	h	s	V	и	h	s
°C		kJ/kg	kJ/kg	kJ/kg K	m <sup>3</sup> /kg	kJ/kg	kJ/kg		m <sup>3</sup> /kg	kJ/kg	kJ/kg	kJ/kg K
	<u>P_ 15.0 l</u>	√Pa (342	.16 C)		P_	1 <u>7.5 MPa</u>	(354.67	(C)	P_	20.0 MPa	<u>a</u> (365.75	
Sat.	0.010341	2455.7	2610.8	5.3108	0.007932	2390.7	2529.5	5.1435	0.005862	2294.8	2412.1	4.9310
350	0.011481		2693.1	5.4438	1							
400	0.015671		2975.7	5.8819	0.012463		2902.4		0.009950	2617.9	2816.9	5.5526
450 500	0.018477 0.020828		3157.9 3310.8	6.1434 6.3480	0.015204 0.017385		3111.4 3276.7		0.012721 0.014793	2807.3 2945.3	3061.7 3241.2	5.9043 6.1446
550	0.020828		3450.4	6.5230	0.017305		3423.6		0.014793	3064.7	3396.2	6.3390
600	0.024921		3583.1	6.6796	0.021073		3561.3		0.018185	3175.3	3539.0	6.5075
650	0.026804		3712.1	6.8233	0.022742		3693.8		0.019695	3281.4	3675.3	6.6593
700	0.028621		3839.1	6.9573	0.024342		3823.5		0.021134	3385.1	3807.8	6.7991
800	0.032121	3609.3	4091.1	7.2037	0.027405	3599.7	4079.3	7.1237	0.023870	3590.1	4067.5	7.0531
900	0.035503		4343.7	7.4288	0.030348		4334.6		0.026484	3795.7	4325.4	7.2829
1000	0.038808		4599.2	7.6378	0.033215		4592.0		0.029020	4004.3	4584.7	7.4950
1100	0.042062		4858.6	7.8339	0.036029	_		7.7588	0.031504	4216.9	4847.0	7.6933
1200 1300	0.045279 0.048469		5122.3 5390.3	8.0192 8.1952	0.038806 0.041556		5117.6 5386.5		0.033952 0.036371	4433.8 4655.2	5112.9 5382.7	7.8802 8.0574
1000	0.040403	P 25.0		0.1002	0.041330	P 30.0		0.1210	0.000071	P_35.		• 0.0374
		_	į		I				1			•
375 400	0.001978 0.006005	1799.9	1849.4 2578.7	4.0345 5.1400	0.001792 0.002798		1791.9 2152.8		0.001701 0.002105	1702.8 1914.9	1762.4 1988.6	3.8724 4.2144
425	0.000003		2805.0	5.4708	0.002790		2611.8		0.002103	2253.3	2373.5	4.7751
450	0.009176		2950.6	5.6759	0.006737		2821.0	5.4422	0.004957	2497.5	2671.0	5.1946
500	0.011143		3165.9	5.9643	0.008691		3084.8	5.7956	0.006933	2755.3	2997.9	5.6331
550	0.012736	3020.8	3339.2	6.1816	0.010175	2974.5	3279.7	6.0403	0.008348	2925.8	3218.0	5.9093
600	0.014140		3493.5	6.3637	0.011445		3446.8		0.009523	3065.6	3399.0	6.1229
650	0.015430		3637.7	6.5243	0.012590		3599.4		0.010565	3190.9	3560.7	6.3030
700	0.016643		3776.0	6.6702	0.013654		3743.9		0.011523	3308.3	3711.6	6.4623
800	0.018922		4043.8	6.9322	0.015628		4020.0	6.8301	0.013278	3531.6	3996.3	6.7409
900	0.021075		4307.1	7.1668	0.017473			7.0695	0.014904	3749.0	4270.6 4541.5	6.9853 7.2069
1000 1100	0.023150 0.025172		4570.2 4835.4	7.3821 7.5825	0.019240 0.020954		4555.8 4823.9		0.016450 0.017942	3965.8 4184.4	4812.4	7.2009 7.4118
1200	0.023172		5103.5	7.7710	0.020334		5094.2		0.017342	4406.1	5085.0	7.6034
1300	0.029115		5375.1	7.9494	0.024279		5367.6		0.020827		5360.2	7.7841
		P_40.0	MPa			P_50.0	MPa			P_60.	0 MPa	
375	0.001641		1742.6	3.8290	0.001560		1716.6		0.001503	1609.7	1699.9	3.7149
400	0.001911		1931.4	4.1145	0.001731		1874.4		0.001633	1745.2	1843.2	3.9317
425 450	0.002538 0.003692		2199.0 2511.8	4.5044 4.9449	0.002009 0.002487		2060.7		0.001816 0.002086		2001.8 2180.2	
500	0.005623		2906.5	5.4744	0.002407			5.1762	0.002000		2570.3	4.9356
550	0.006985		3154.4	5.7857	0.005118		3025.4		0.003955		2901.9	5.3517
600	0.008089		3350.4	6.0170	0.006108		3252.6		0.004833		3156.8	5.6527
650	0.009053		3521.6	6.2078	0.006957		3443.5		0.005591	3031.3	3366.8	5.8867
700	0.009930		3679.2	6.3740	0.007717	3228.7	3614.6	6.2179	0.006265		3551.3	6.0814
800	0.011521	3511.8	3972.6	6.6613	0.009073		3925.8		0.007456	3432.6	3880.0	6.4033
900	0.012980		4252.5	6.9107	0.010296		4216.8		0.008519		4182.1	6.6725
1000	0.014360		4527.3	7.1355	0.011441		4499.4		0.009504		4472.2	
1100	0.015686		4801.1	7.3425	0.012534		4778.9		0.010439		4757.3	7.1255
1200	0.016976		5075.9	7.5357 7.7175	0.013590		5058.1		0.011339			7.3248
1300	0.018239	40∠3.3 •	5352.8	7.7175	0.014620	4007.5	5338.5	7.0046	0.012213	4091.0	0024.0	1.0111 •

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